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Japanese Kokai Patent Application No. Sho 61[1986]-25763

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WORKPIECE HOLDING MECHANISM FOR A PLANE POLISHING DEVICE

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[There are no amendments to this patent.]

Claim

1. A workpiece holding mechanism for a plane polishing device characterized in that it contains a holding part, which holds a workpiece that is on top of the polishing surface of the plane polishing device and is equipped with a convex spherical surface centering around one point on the surface of the said workpiece to be processed; a support part, which is provided for maintaining a constant orientation and is equipped with a concave spherical surface, which engages with the aforementioned convex spherical surface in a freely oscillating manner centering about one point on the aforementioned workpiece; and a flexible body, which is provided between the aforementioned holding part and the aforementioned supporting part and has high torsional rigidity but can bend freely.

Detailed explanation of the invention

Industrial application field

The present invention concerns a workpiece holding mechanism for a plane polishing device. In particular, it concerns a holding mechanism for a workpiece in a plane polishing device which polishes the surface of thin plates.

Prior art

Generally, a workpiece holding mechanism for a plane polishing device is constructed to include a holding area, where the workpiece is held on top of a polishing surface of the plane polishing device, and the surface of the workpiece is polished by oscillating [vibrating while moving] the workpiece over the polishing surface.

Figure 2 is a longitudinal section of a workpiece holding mechanism for a plane polishing device of the prior art. In Figure 2, a disk (1) of the plane polishing device is rotated about a shaft (2). Also, a sleeve (4) is attached to a frame (3) of the plane polishing device in a freely rotatable manner, and a splined shaft (5) is attached to the central hole of this sleeve (4) in a freely movable manner in the direction of the shaft and in such a manner that it rotates together with the sleeve (4) about the shaft. A lever (7), which is attached to an air cylinder (6) provided for the frame (3), engages with the splined shaft (5) in a freely rotatable manner. Also, a gear (9), which is attached to a motor (8) provided for the frame (3), engages with a gear (10), which is provided at the sleeve (4).

A hemispherical body (11) engages with the curved area in the form of a spherical surface that is provided at the front end of the splined shaft (5) in a freely oscillating manner. A pressing plate (12) is fixed to the hemispherical body (11), and a frame (13) is provided at the pressing plate (12). A pin (14), which is provided at the frame (13), engages with a groove (15), which is provided at the front end of the splined shaft (5). A compression spring (16), which is provided between the splined shaft (5) and the frame (13), interacts to press the hemispherical body (11) onto the splined shaft (5) in order to prevent the hemispherical body (11) from falling when the splined shaft (5) ascends.

A through-hole (17), which is provided at the pressing plate (12), and a through-hole (18), which is provided between the hemispherical body (11) and the pressing plate (12), are connected to a vacuum pump (not shown) through a pipe (19), which passes through a hole provided at the splined shaft (31), in order to vacuum hold material (20), which is a magnetic disk forming the workpiece, onto the pressing plate (12). A ring (21) is also provided and fixed at the pressing plate (12) in order to determine the position of the material (20).

To polish the surface of the material (20) with this plane polishing device, the air cylinder (6) is actuated so that the pressing plate (12) ascends and so that the material (20) is vacuum held against the inner side of the ring (21) at the lower surface of the pressing plate (12). Next, the pressing plate (12) is lowered by the air cylinder (6) through rotation of the motor (8), and the material (20) is pressed against the polishing surface (22) of the disk (1). Also, a polishing solution (not shown) is spread over the polishing surface (22). Accordingly,

the bottom surface of the material (20) is polished by the action of its own rotations and vibrations by the rotation of the disk (1).

The polishing surface (22) of the disk (1) is processed to have a flat surface; however, a small amount of waviness remains in many actual cases. Accordingly, it is necessary for the material (20) and the pressing plate (12) to be able to tilt slightly along the waviness of the polishing surface (22) in order for the material (20) constantly to adhere close to the polishing surface (22) for a smooth finish. This tilting is obtained when the hemispherical body (11) vibrates with the spherical concave area of the splined shaft (5). Moreover, the material (20) tilts while centering around the center C because the center C of the spherical surface of the hemispherical body (11) is established to be positioned at the bottom surface of the material (20), and the position of the bottom surface of the material (20) does not change even though it is tilted, and polishing can occur.

The pipe (19) is elastic and can absorb some tilting in the hemispherical body (11). Also, the hemispherical body (11) vibrates around the splined shaft (5); therefore, it is designed so that the rotation by the motor (8) is transmitted to the pressing plate (12) and the material (20) when the pin (14) engages with the groove (15).

Problems to be solved by the invention

However, the ability of the pressing plate (12) and material (20) to follow the waviness of the polishing surface (22) was not satisfactory. One factor is the generation of a large amount of

friction between the pin (14) and the groove (15). Figure 3 is a schematic diagram explaining the force that is applied to the pin (14), and it corresponds to the right side surface diagram of the major part in Figure 2. In Figure 3, force b, which is equal to the friction between the material (20) and the polishing surface (22), is applied to the groove (15) from the pin (14) when the splined shaft (5) rotates, as illustrated by arrow a.

Furthermore, since a condition is created, in which the right side opens between the material (20) and the polishing surface (22), as illustrated in Figure 1, by the waviness of the polishing surface (22), and if force P is obtained by the piston (6), force P interacts upwards at the left edge of the material (20). To consider the equilibrium of the moment about center C, where the length between center C of the spherical surface of the hemispherical body (11) and the left edge of the material (20) is d and the height between center C and the pin (14) is h, a force of Pd/h is also applied to the pin (14). In practice, this force P becomes considerably large; therefore, a large force also acts on the pin (14), resulting in a large frictional force.

There was also the problem of the pin (14) being constantly pressed toward the left by the groove (15) in Figure 3, causing the pressing plate (12) to swing around the pin (14) according to the waviness of the polishing surface (22), the base ([illegible]) of the pin (14) to change its position to the left or the right relative to the splined shaft (5), and a fluctuation to occur in the rotation of the pressing plate (12).

The aim of the present invention is to offer a workpiece holding mechanism for a plane polishing device in which the aforementioned problems are solved, there is a satisfactory following of the waviness of the polishing surface by tilting of

the workpiece, and the fluctuation in the rotation of the workpiece is reduced for a smooth polishing of the workpiece.

Means to solve the problems .

The present invention comprises a holding part (34), which holds the workpiece that is on top of the polishing surface of the plane polishing device and is equipped with a convex spherical surface centering around one point on the surface of the said workpiece to be processed; a support part (32), which is provided for maintaining a constant orientation and is equipped with a concave spherical surface, which engages with the aforementioned convex spherical face in a freely vibrating manner centering about one point on the aforementioned workpiece; and a flexible body (36), which is provided between the aforementioned holding part (34) and the aforementioned support part (32) and has high torsion rigidity but can bend freely.

Function

The elastic body (36), which has torsional rigidity but can bend freely, tilts the holding part (34) against the support part (32) while following the waviness of the polishing surface and not generating a large amount of friction. During this process, the holding part (34) does not separate from the support part (32) in the direction of rotation.

Application example

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Next, an application example of the present invention will be explained with reference to a figure. Figure 3 [sic; 1] is a longitudinal section of an application example of the present invention. A disk (1), shaft (2), frame (3), sleeve (4), air cylinder (6), lever (7), motor (8), and gears (9) and (10) are the same as those illustrated in Figure 1 [sic; 2]. A splined shaft (31) is attached to the sleeve (4) so that it can freely oscillate in the direction of the shaft and rotate together with it around the shaft. A hemispherical body (33) engages with the concave part in the form of a spherical surface, which is provided at a flange (32) at the lower end of the splined shaft (31) in a freely oscillating manner. A pressing plate (34) is fixed to the hemispherical body (33). A through-hole (35) of the pressing plate (34) is connected to a pipe (19) in order to hold the material (20) against the pressing plate (34).

The upper end of bellows (36) is fixed to the flange (32) and its lower end to the pressing plate (34). The torsional rigidity of the bellows (36) with respect to the central shaft is high, but it can expand and bend in the direction of the central shaft; therefore, the pressing plate (34) does not separate from the flange (32) in the direction of rotation, but it can tilt freely. Accordingly, a large frictional force is not generated even when the pressing plate (34) is tilted, and the pressing plate (12) and the material (20) satisfactorily follow the waviness of the polishing surface.

The present invention can also be applied to plane polishing devices, in which the disk is fixed, and the pressing plate (34),

for example, rotates together with the frame (3) around the shaft (12).

A steel ball, for example, may also be included between the concave spherical surface of the supporting part and the convex spherical surface of the holding part so that the friction can be reduced.

Furthermore, the elastic body that is provided between the support part and the holding part does not necessarily have the form of a bellows. For example, dividing the bellows in the circumferential direction, in other words, several plate springs that are bent in the middle and arranged over the circumference may also be used.

Effect of the invention

As explained above, in the workpiece holding mechanism for a plane polishing device of the present invention, the holding part is tilted without the generation of a large amount of friction between the groove and the pin by using an elastic body which has torsional rigidity but which can expand and bend freely, instead of an engagement between the groove and the pin, and the workpiece can satisfactorily tilt with and follow the waviness of the polishing surface.

Also, oscillations around the pin are eliminated when the support part is tilted, a fluctuation in the rotating speed of the workpiece can be made very small, and the effect is smooth polishing of the workpiece.

Brief description of the figures

Figure 1 is a longitudinal section of an application example of the present invention. Figure 2 is a longitudinal section of an example of a workpiece holding mechanism for a plane polishing device of the prior art. Figure 3 is a model diagram which explains the force which interacts on the pin (14) as an example illustrated in Figure 2.

1...disk, 5, 31...splined shaft, 11, 33...hemispherical body, 12, 34...pressing plate, 14...pin, 15...groove, 20...material, and 36...bellows.

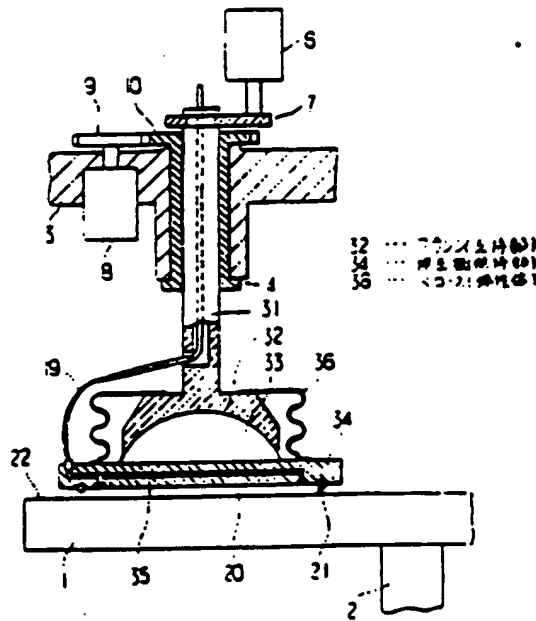


Figure 1

- Key: 32 Flange (supporting part)
 34 Pressing plate (holding part)
 36 Bellows (elastic body)

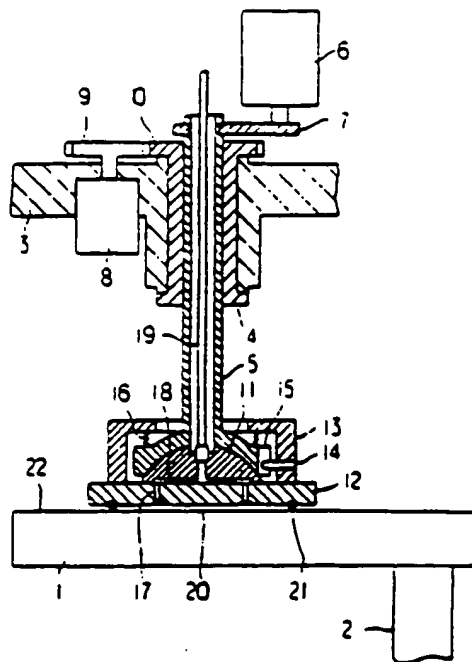


Figure 2

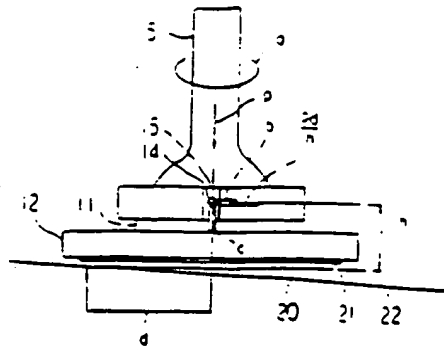


Figure 3

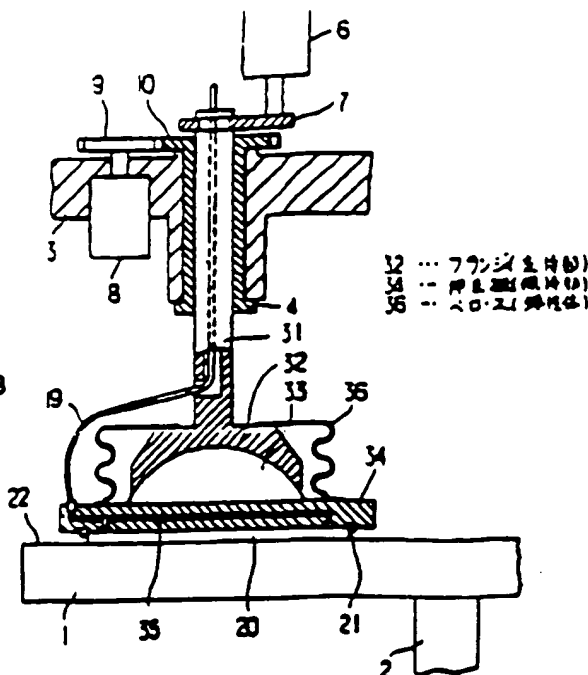
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INVENTOR : KAMATA TAKEMI; others: 01

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 B24B41/06

TITLE : WORK HOLDING MECHANISM FOR
 SURFACE POLISHING MACHINE



ABSTRACT : PURPOSE: To polish a work smoothly by providing a resilient bellows between the work holding section having convex face and the supporting member having concave face engagable slidably with the convex face.
 CONSTITUTION: Semi-spherical body 33 secured to a pressboard 34 is engaged slidably with spherical recess made in the lower end flange 32 of spline shaft 31 to adsorb a material 20 through a hole 35 communicated with a tube 19 to the pressboard 34. A bellows 36 having high rigidity in the rotary direction while flexible against the vertical shrinkage and bending is secured between said flange 32 and the pressboard 34. Consequently, the work 20 or the pressboard 34 will follow the waving of the polishing face 22 well to reduce the fluctuation of the rotary speed of the work 20 thus to polish the work 20 smoothly.

① 許出公開

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識別記号

戸内整理番号

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41/08

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8308-3C

審査請求 未請求 発明の数 1 (全4頁)

④発明の名称 平面研摩装置の被加工物保持機構

① 昭59-145408

出 版 昭59(1984)7月13日

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代理人 丹瑞士 曾 野 中

● ● ●

1. 発明の名称

平 湖 供 應 總 局 工 務 工 務 保 障 處

2. 世界需求の概観

(1) 平面図形面積の算術上の増加工物を保持しこの増加工物の増加工面上の一点を中心とする三角形を有する保持部と、保持部を一定に保つて置かれ、増記増加工面上の一点を中心とし増記三角形の面積が元の面積と等しい三角形を有する支持部と、増記保持部と増記支持部の間に置かれ張り代としては割増が小さく割代としてはある数を保持部とを有することを特徴とする平面図形面積の増加工物保持構造。

1 愚明の形相 2 愚明

： 電 氣 工 事 分 野 ；

を説明は、イ造研鑽機盤力の加工工物材料特性、
ウ造研鑽機の構造を研究する凡此のイ造研鑽機盤力の
加工工物材料特性に關する。

(或 電 價 的)

一般に、この種の建築費の増大は、建設費の増大に比例する。平

石油の精製の研究以上の石油工業を促進する機関
部を含んで組織され、研究員上において皆等しく
を組織させて労働工場の発展を促進している。

第三圖は、従来の平野伊勢組の首長二羽目伊
佐雄の周回集團である。男を認めていて平野伊勢
組の内幕には男を中心として組織が作られる。
一方平野伊勢組のメンバーには女も入るが
組織を生か取り付かれ、このメンバーの中心
でヒューマン・リレーションが確立されて組織作り
にメンバーと一線をとって行動するようになり
付けられている。メンバーは設計されたメンバ
ー・システムを取り付けたという状態にある。
取り付けたメンバーは多量に入っている。また、
設計したメンバーに取り付けられなかったメン
バーは組織外とみなされている。

「アタラン」船ものも同様に船隻数の減少に
影響され、船数内に収め合っている。4月18日に
阿波連は同船とされ、阿波連12月、4月13日
にされている。阿波連は同船とされ、阿波連12月、
4月13日にされている。阿波連は同船とされ、阿波連12月、
4月13日にされている。

戸原保12は投げられた通風17及び本機体11と戸原保12の間で投げられた通風18は、エアライノ細31で投げた穴を通過する習19を介して風量ポンプ（表示4時）に通過され、排気口である通風4エアの裏面20を戸原保18が風量ポンプするたのりものである。また風量20の位置を定めるたのり戸原保12はリノ細31が通過されている。

この平面図解法で、図形20の真面を投影するに
は、エアシリンダを移動させて中心線18を上昇
させ、図形20を中心線18の下置の5/1621の内側
に位置させる。スカラーノミより距離を量
ながら中心線18をエアシリンダノミより下移させ
図形20を円盤1の投影面に投影し行々。また型
式は示してないが投影面23は、投影面が数有
れている。従つて図形20の下置は、自からの距離
及び円盤1の面線による移動で投影される。

にもたつてゐる。第3型は、ピン14に作用する力を説明するたのの図で、第3型の主要部の右側面図に相当する。第3型の矢印は、支承するようによアライニング軸が回転してあれば、最初と終極面23との摩擦力に釣り合う力がピン14により面15に与えられる。さらに終極面23のうねりにより最初と終極面23との摩擦第3型に支承するようによ右側が同じ状態をつたへし、ピン16により力Fが与えられるとすると、最初との摩擦に向上する力Fが働く。半球体11の表面の中心Cと最初との接触までの距離を、中心Cからピン16までの距離をとり、中心Cをわりのモーメントの釣合を考へるとピン16は $\frac{Pd}{h}$ の力が作用する。最終的な力Fがかなり大きくなるためピン16も大きな力が作用したと見做す方がよい。

また、第3期において、ビン16は通13日より高反りとなり、足元も厚みにもことなり、新巻面は、うなりが成つて、押込部12は、ビン16を中心として、張り固めすることとなり、エプラインも、成り、して、増す、的、ビン16の増え、は、新巻面、成り、し、押込部12の、

調査した研究船22は、研究船であるが、大半はこぼれてへらが資源的には温かであるがうなりが鳴まれている船が多い。又つて東門20を研究船22に曳き寄せて明らかに研究するときは、東門20は少戸年産12を研究船22のうなりで促つて多少傾くことが出来るようにすると望みがある。この船は、半線11のメソフ・ン船との船体の図面との所載で明らか、しかも半線11の船体の中心が東門20の下層に位置するように設置されているもので、東門20は中心を中心として船身、船頭も東門20の下層の位置は変化せず研究することが出来る。

たか晋18に低位を有し平陸部11の多少の傾きは
後傾である。また平陸部11がユノラインを越えて西
し屈曲するため、ピン14と曲15の組合せより押陸
部12及び曲部20までサークもたよる面転が起わる
ようにしている。

(愚明か同義しゝうとすも思明也)

しかし、戸生 12 及び 10 の研鑽を 11 のうた
りて研する通説性はまだよくをかつた。この研
鑽の一つは、14 と 15 の間を、なりを置き、力を入

て置かれた。それによつて、

を説明の目的で、上記欠点を除去し、加工工物の値目の明細表のうねりや円滑な連続性がよく、また加工工物の製造変動を少くして円滑に加工工物を製造することが可能で平準生産設備の加工工物使用時間を短縮することにある。

(問題点を解決するための手段)

本発明は、平面投影曲線の保線層上の加工工物を保持しつゝ加工工物の加工面上の一点を中心とする△分岐を有する偏周部と、保線を一定に保つて居り且つ記録加工物上の一点を中心とし所記△分岐に明確目印を保存する凹部面を有する主閉鎖部と、上記凹部面と前記主閉鎖部との間に設けられ且つ互に對して該凹部が深くかつ鋭角として形成される傾斜部とを有するものである。

134

[illegible]

押し回転方向はすれをい。

(実施例)

次に本発明の実施例について図面を参照して説明する。第1図は本発明の一実施例の縦断面図である。円盤1、軸2、フレーム3、スリーブ4、エアシリンダ5、レバー7、ネジ8、歯車9,10は第1図に示すものと同じである。スプライン軸31は、軸方向には歯車9及び歯車10の中心軸と一致し、かつ回転するようスリーブ4に取り付けられている。スプライン軸31の下端のフランジ32に設けた歯車33の歯部が歯車9の歯部と係合している。歯車33は押圧部34が設けられている。押圧部34の中心孔35は管19と連結され、材料を押圧部34に供給するためのものである。

ベローズ36が上端をフランジ32に固定し下端を押圧部34に固定して設けられている。ベローズ36は中心軸周りのねじりに対しては剛性が大きい。すなわち、中心軸方向の伸縮及び曲げに対しては柔軟であるため、押圧部34はフランジ32に対して軸方向にすれをい、しかも自由に動くことが

できる。従って押圧部34がねじるときも大きな力を生じて押圧部34及び管20の研削面を削削する適成性はよい。

本発明は、円盤が固定してあつてフレーム3とともに押圧部34が軸2を中心として回転するように平面研削装置にも適用できる。

また支持部と保持部との間の歯車の噛み合部を介して、摩擦力を減少させることもできる。

さらに支持部と保持部との間に設ける弾性は、必ずしもベローズの形状をしている必要はない。例えばベローズを断面方向に分割したもの、言い換えれば中間を等間隔に設けた複数の板状のものを円周上に並べたものでもよい。

(発明の効果)

本発明の平面研削装置の被加工物保持機構は、以上説明したように歯とベンの係合の代わりねじりに対しては剛性を有し伸縮及び曲げに対しては柔軟性のある弾性体を用いることにより、歯とベンの間の大きな摩擦力を発生させることなく保持部

がねじ、研削面のうねりに対する被加工物の軸心の適成性をよくすることが出来る。

また保持部がねじるときにベンを中心として回転することができ、被加工物の回転速度の調節を容易にすることができる。円盤に被加工物を研削できる効果がある。

4. 図面の簡単な説明

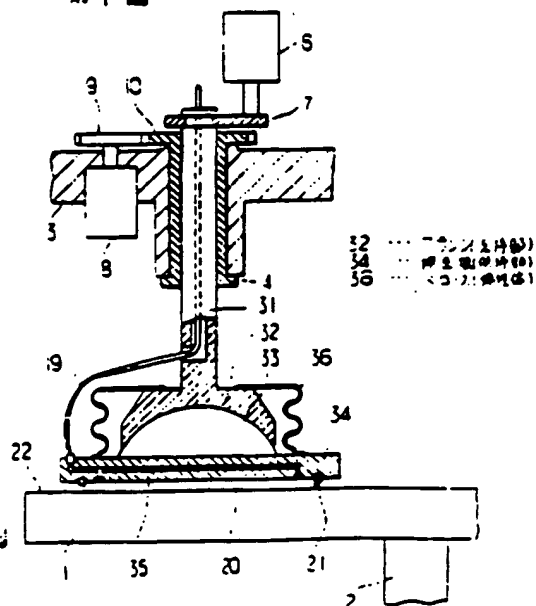
第1図は本発明の一実施例の縦断面図、第2図は平面研削装置の被加工物保持機構の概略図、第3図は第1図に示すベンのベネ14に作用する力を説明するための模式図である。

1 円盤、2 軸、3, 31 スプライン軸、11, 33 歯車、12, 34 押圧部、14 ベン、15 歯、20 管、30 ベローズ。

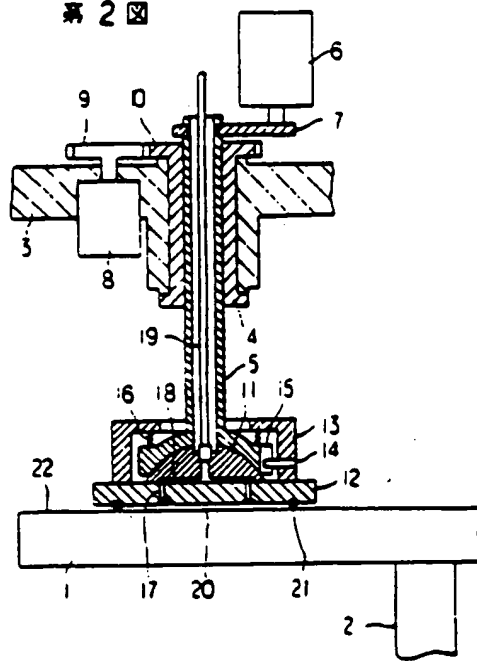
特許代理人 日本電気株式会社

代理人 大塚 啓 野 中

第1図



第 2 图



第 3 图

